

ภาคผนวก ง

โปรแกรมที่ใช้ในการคำนวณ

โปรแกรมที่ใช้ในการคำนวณ (ปรับปรุงจากโปรแกรมของ Simple ของ Urieli)
 ในส่วนนี้แสดง Source Code ของโปรแกรม MATLAB ที่ใช้ในการคำนวณ

ง1 โปรแกรมคำนวณกำลังบังคับเครื่องยนต์ GPU-3

โปรแกรมเพื่อคำนวณหากำลังและความร้อนที่ให้กับเครื่องยนต์ ปรับปรุงจากโปรแกรม simple ของ Urieli ประกอบด้วยไฟล์

spec	ข้อมูลจำเพาะของเครื่องยนต์
cycle.m	ไฟล์ที่กำหนด อุณหภูมิและความเร็วรอบ เรียกการทำงานของโปรแกรมหลัก
sea.m	โปรแกรมหลัก กำหนดตัวแปร และเรียกโปรแกรมการคำนวณ
define.m	อ่านข้อมูลจากไฟล์ข้อมูลจำเพาะ กำหนดค่าจากข้อมูลจำเพาะให้กับตัวแปร คำนวณหาอุณหภูมิ
engine.m	กำหนดข้อมูลรูปแบบของเครื่องยนต์
heatex.m	กำหนดตัวแปรขนาดของ คูลเลอร์
regen	ตัวแปรขนาดของ ฮีทเตอร์ คูลเลอร์ และรีเจนเนอเรเตอร์
gas.m	กำหนดสารทำงานที่ใช้ และคำนวณหาค่าสมบัติของก๊าซ เช่น ความจุความร้อน จำเพาะ ความหนืดจลน์
operat.m	คำนวณมวลสารทำงานและทำการวิเคราะห์แบบ Schmidt
simple.m	วิเคราะห์ด้วยแบบจำลอง simple ซึ่งรวมผลของการถ่ายเทความร้อนและความดัน ลดในฮีทเตอร์ รีเจนเนอเรเตอร์ และคูลเลอร์
adiab.m	วิเคราะห์ด้วยแบบจำลองอะเดียบาติกและแสดงผลการคำนวณ
dadiab.m	กำหนดสมการอนุพันธ์ที่จะหาคำตอบด้วยวิธีรุงเงคุททาคันดับสี่
volume.m	คำนวณหาการเปลี่ยนแปลงปริมาตรในแต่ละมุมหมุนของเพลลา
rk4.m	คำนวณด้วยวิธีรุงเงคุททาคันดับสี่
regsim.m	คำนวณผลและประสิทธิภาพของรีเจนเนอเรเตอร์
worksim.m	คำนวณผลของความดันลดและงานที่สูญเสีย

reynum.m คำนวณเรย์โนลด์์นัมเบอร์ในฮีทเตอร์และคูลเลอร์
reynum_t คำนวณเรย์โนลด์์นัมเบอร์ในรีเจนเนอเรเตอร์

spec

```
s
8.724e-006
7.757e-005
1.349e-005
8.117e-005
120.0
p
1.016e-003
0.066
152
t
2.082e-003
2.032e-003
2.032e-003
80
m
0.690
4.000e-005
p
3.020e-003
0.2453
96
he
10200000.0
349.0
933.0
50.0
```

cycle.m

```
clc;
clear all;
global res
global temp_h temp_k
global freq_h

temp=[965 968 969];
tempk=[337 347 353];
rpm=[41.66 50 58.33];

for(i = 1:1:3)

    temp_h=temp(i);
    temp_k=tempk(i);
    freq_h=rpm(i);
    sea(temp_h,temp_k,freq_h);
end
```

sea.m

```

% sea (stirling engine analysis) - main program
%Israel Urieli 7/20/02

function sea(temp_h,temp_q,freq_h)
% Row indices of the var, dvar arrays:
TC = 1; % Compression space temperature (K)
TE = 2; % Expansion space temperature (K)
QK = 3; % Heat transferred to the cooler (J)
QR = 4; % Heat transferred to the regenerator (J)
QH = 5; % Heat transferred to the heater (J)
WC = 6; % Work done by the compression space (J)
WE = 7; % Work done by the expansion space (J)
W = 8; % Total work done (WC + WE) (J)
P = 9; % Pressure (Pa)
VC = 10; % Compression space volume (m^3)
VE = 11; % Expansion space volume (m^3)
MC = 12; % Mass of gas in the compression space (kg)
MK = 13; % Mass of gas in the cooler (kg)
MR = 14; % Mass of gas in the regenerator (kg)
MH = 15; % Mass of gas in the heater (kg)
ME = 16; % Mass of gas in the expansion space (kg)
TCK = 17; %Conditional temperature compression space / cooler (K)
THE = 18; %Conditional temeprature heater / expansion space (K)
GACK = 19;%Conditional mass flow compression space/cooler (kg/rad)
GAKR = 20; % Conditional mass flow cooler / regenerator (kg/rad)
GARH = 21; % Conditional mass flow regenerator / heater (kg/rad)
GAHE = 22; % Conditional mass flow heater / expansion space (kg/rad)
ROWV = 22; % number of rows in the var matrix
ROWD = 16; % number of rows in the dvar matrix
COL = 37; % number of columns in the matrices (every 10 degrees)
%=====
global tk tr th % cooler, regenerator, heater temperatures [K]
global vk % cooler void volume [m^3]
global vr % regen void volume [m^3]
global vh % heater void volume [m^3]
global res
global temp_h temp_k
global freq_h
define(temp_h,temp_k,freq_h);

res = fopen('filedata','a');
[var,dvar] = simple(res);
fprintf('quitting simulation...\n');
status = fclose(res);

```

engine.m

```

function engine
% Define engine configuration and drive geometric parameters.
% Israel Urieli 4/14/02

global engine_type % s)inusoidal, y)oke (both alpha engines)
global new fid % new data file

engine_type = 'u';

```

```

while(strncmp(engine_type,'u',1))
    if(strncmp(new,'y',1))
        fprintf('Available engine types are:\n');
        fprintf('  s)inusoidal drive\n');
        fprintf('  y)oke drive (Ross)\n');
        engine_type = input('enter engine type ','s');
        fprintf(fid, '%c\n', engine_type(1));
    else
        engine_type = fscanf(fid, '%c',1);
    end
    if(strncmp(engine_type,'s',1))
        sindrive;
    else
        fprintf('engine type is undefined\n')
        engine_type = 'u';
    end
end
end
=====
function sindrive
% Sinusoidal drive engine configuration
% Israel Urieli 4/14/02

global vclc vcle % compression,expansion clearence vols [m^3]
global vswc vswe % compression, expansion swept volumes [m^3]
global alpha % phase angle advance of expansion space [radians]
global new fid % new data file

fprintf('sinusoidal drive engine configuration\n')
if(strncmp(new,'y',1))
    vclc = input('enter compression space clearence volume[m^3]: ');
    vswc = input('enter compression space swept volume [m^3]: ');
    vcle = input('enter expansion space clearence volume [m^3]: ');
    vswe = input('enter expansion space swept volume [m^3]: ');
    phase =input('enter expansion phase angle advance [degrees]: ');
    fprintf(fid, '%.3e\n', vclc);
    fprintf(fid, '%.3e\n', vswc);
    fprintf(fid, '%.3e\n', vcle);
    fprintf(fid, '%.3e\n', vswe);
    fprintf(fid, '%.1f\n', phase);
else
    vclc = fscanf(fid,'%e',1);
    vswc = fscanf(fid,'%e',1);
    vcle = fscanf(fid,'%e',1);
    vswe = fscanf(fid,'%e',1);
    phase = fscanf(fid, '%f',1);
end
end
fprintf('\nsinusoidal drive engine data summary:\n');
fprintf('comp clearence,swept vols%.1f,%.1f[cm^3]\n',
vclc*1e6,vswc*1e6);
fprintf(' exp clearence,swept vols %.1f,%.1f[cm^3]\n',
vcle*1e6,vswe*1e6);
fprintf(' expansion phase angle advance %.1f[degrees]\n', phase);
alpha = phase * pi/180;
=====

```

heatex.m

```

function heatex
% Specify heat exchanger geometric parameters
% Jeff Guess 1/26/03
cooler;
regen;
heater;

%=====
function cooler
% Specify cooler geometric parameters
% Jeff Guess 1/26/03
global vk % cooler void volume [m^3]
global ak % cooler internal free flow area [m^2]
global awgk % cooler internal wetted area [m^2]
global dk % cooler hydraulic diameter [m]
global lk % cooler effective length [m]
global new fid % new data file

cooler_type = 'u';
while(strncmp(cooler_type,'u',1))
    if(strncmp(new,'y',1))
        fprintf('Available cooler types are:\n')
        fprintf('  p, for smooth pipes\n')
        fprintf('  a, for smooth annulus\n')
        fprintf('  s, for slots\n')
        cooler_type = input('enter cooler type ','s');
        fprintf(fid, '%c\n', cooler_type(1));
    else
        fscanf(fid, '%c',1);
        cooler_type = fscanf(fid, '%c',1);
    end
    if(strncmp(cooler_type,'p',1))
        [vk,ak,awgk,dk,lk] = pipes;
    elseif(strncmp(cooler_type,'a',1))
        [vk,ak,awgk,dk,lk] = annulus;
    elseif(strncmp(cooler_type,'s',1))
        [vk,ak,awgk,dk,lk] = slots;
    else
        fprintf('cooler type is undefined\n')
        cooler_type = 'u';
    end
end

fprintf('cooler data summary:\n');
fprintf(' void volume(cc) %.4f\n', vk*1e6)
fprintf(' free flow area (cm^2) %.4f\n', ak*1e2)
fprintf(' wetted area (cm^2) %.4f\n', awgk*1e2)
fprintf(' hydraulic diameter(mm) %.4f\n', dk*1e3)
fprintf(' cooler length (cm) %.4f\n', lk*1e2)

%=====

```

heater.m

```

function heater
% Specify heater geometric parameters
% Israel Urieli 4/15/02

```

```

global vh % heater void volume [m^3]
global ah % heater internal free flow area [m^2]
global awgh % heater internal wetted area [m^2]
global dh % heater hydraulic diameter [m]
global lh % heater effective length [m]
global new fid % new data file

heater_type = 'u';
while(strncmp(heater_type,'u',1))
    if(strncmp(new,'y',1))
        fprintf('Available heater types are:\n')
        fprintf('  p, for smooth pipes\n')
        fprintf('  a, for smooth annulus\n')
        fprintf('  s, for slots\n')
        heater_type = input('enter heater type ','s');
        fprintf(fid, '%c\n', heater_type(1));
    else
        fscanf(fid, '%c',1); % bypass the previous newline
character
        heater_type = fscanf(fid, '%c',1);
    end
    if(strncmp(heater_type,'p',1))
        [vh,ah,awgh,dh,lh] = pipes;
        vh=80.84e-6; % for GPU3
    elseif(strncmp(heater_type,'a',1))
        [vh,ah,awgh,dh,lh] = annulus;
    elseif(strncmp(heater_type,'s',1))
        [vh,ah,awgh,dh,lh] = slots;
    else
        fprintf('heater type is undefined\n')
        heater_type = 'u';
    end
end
fprintf('heater data summary:\n');
fprintf(' void volume(cc) %.2f\n', vh*1e6)
fprintf(' free flow area (cm^2) %.2f\n', ah*1e2)
fprintf(' wetted area (cm^2) %.2f\n', awgh*1e2)
fprintf(' hydraulic diameter(mm) %.2f\n', dh*1e3)
fprintf(' heater length (cm) %.2f\n', lh*1e2)

%=====
function [v,a,awg,d,len] = pipes
% homogeneous smooth pipes heat exchanger
% Israel Urieli 4/15/02
global new fid % new data file

fprintf('homogeneous bundle of smooth pipes\n')
if(strncmp(new,'y',1))
    d = input('enter pipe inside diameter [m] : ');
    len = input('enter heat exchanger length [m] : ');
    num = input('enter number of pipes in bundle : ');
    fprintf(fid, '%.3e\n', d);
    fprintf(fid, '%.3f\n', len);
    fprintf(fid, '%d\n', num);
else
    d = fscanf(fid,'%e',1);
    len = fscanf(fid,'%f',1);

```

```

        num = fscanf(fid,'%d',1);
    end
    a = num*pi*d*d/4; %pi/4*d^2
    v = a*len;
    awg = num*pi*d*len; %wall/gas area
    %=====
function [v,a,awg,d,len] = annulus
% annular gap heat exchanger
% Israel Urieli 4/15/02
global new fid % new data file

fprintf(' annular gap heat exchanger\n')
if(strncmp(new,'y',1))
    dout = input('enter annular gap outer diameter [m] : ');
    din = input('enter annular gap inner diameter [m] : ');
    len = input('enter heat exchanger length [m] : ');
    fprintf(fid, '%.3f\n', dout);
    fprintf(fid, '%.3f\n', din);
    fprintf(fid, '%.3f\n', len);
else
    dout = fscanf(fid,'%f',1);
    din = fscanf(fid,'%f',1);
    len = fscanf(fid,'%f',1);
end

a = pi*(dout*dout - din*din)/4;
v = a*len;
awg = pi*(din + dout)*len;
d = dout - din;
%=====
function [v,a,awg,d,len] = slots
% slots heat exchanger
% Israel Urieli 3/31/02
global new fid % new data file

fprintf(' slots heat exchanger\n')
if(strncmp(new,'y',1))
    w = input('enter width of slot [m] : ');
    h = input('enter height of slot [m] : ');
    len = input('enter heat exchanger length [m] : ');
    num = input('enter number of slots : ');
    fprintf(fid, '%.3e\n', w);
    fprintf(fid, '%.3e\n', h);
    fprintf(fid, '%.3f\n', len);
    fprintf(fid, '%d\n', num);
else
    w = fscanf(fid,'%f',1);
    h = fscanf(fid,'%f',1);
    len = fscanf(fid,'%f',1);
    num = fscanf(fid,'%d',1);
end

a = num*w*h;
v = a*len;
awg = num*2*(w + h)*len;
d = 4*v/awg;
%=====

```

regen.m

```

function regen
% Specifies regenerator geometric and thermal properties
% Israel Urieli 04/20/02

global lr % regenerator effective length [m]
global cqwr % regenerator housing thermal conductance [W/K]
global new fid % new data file

regen_type = 'u';
while(strncmp(regen_type,'u',1))
    if(strncmp(new,'y',1))
        fprintf('Available regenerator configurations are:\n')
        fprintf('  t, for tubular regenerator set\n')
        fprintf('  a, for annular regenerator\n')
        regen_type = input('enter regenerator configuration','s');
        fprintf(fid, '%c\n', regen_type(1));
    else
        fscanf(fid, '%c',1); % bypass the previous newline character
        regen_type = fscanf(fid, '%c',1);
    end
    if(strncmp(regen_type,'t',1))
        fprintf('tubular regenerator housing\n')
        if(strncmp(new,'y',1))
            dout = input('enter tube housing external diameter [m] : ');
            din = input('enter tube housing internal diameter [m] : ');
            lr = input('enter regenerator length [m] : ');
            num = input('enter number of tubes : ');
            fprintf(fid, '%.3e\n', dout);
            fprintf(fid, '%.3e\n', din);
            fprintf(fid, '%.3e\n', lr);
            fprintf(fid, '%d\n', num);
        else
            dout = fscanf(fid, '%f',1);
            din = fscanf(fid, '%f',1);
            lr = fscanf(fid, '%f',1);
            num = fscanf(fid, '%d',1);
        end
        dimat = 0;

    elseif(strncmp(regen_type,'a',1))
        fprintf('annular regenerator housing\n')
        if(strncmp(new,'y',1))
            dout = input('enter housing external diameter [m] : ');
            din = input('enter housing internal diameter [m] : ');
            dimat = input('enter matrix internal diameter [m] : ');
            lr = input('enter regenerator length [m] : ');
            fprintf(fid, '%.3e\n', dout);
            fprintf(fid, '%.3e\n', din);
            fprintf(fid, '%.3e\n', dimat);
            fprintf(fid, '%.3e\n', lr);
        else
            dout = fscanf(fid, '%f',1);
            din = fscanf(fid, '%f',1);
            dimat = fscanf(fid, '%f',1);
            lr = fscanf(fid, '%f',1);
        end
    end
end

```

```

        end
        num = 1;
    else
        fprintf('regenerator configuration is undefined\n')
        regen_type = 'u';
    end
end
end

amat = num*pi*(din*din - dimat*dimat)/4; % regen matrix area
awr = num*pi*(dout*dout - din*din)/4; % regen housing wall area
kwr = 19; % thermal conductivity [W/m/K]
cqwr = kwr*awr/lr; % regen wall thermal conductance [W/K]
matrix(amat);

%=====
function matrix(amat)
% Specifies regenerator matrix geometric and thermal properties
% Israel Urieli 03/31/02

global matrix_type % m)esh or f)oil
global new fid % new data file

matrix_type = 'u';
while(strncmp(matrix_type,'u',1))
    if(strncmp(new,'y',1))
        fprintf('Available matrix types are:\n')
        fprintf('  m, for mesh matrix\n')
        fprintf('  f, for foil matrix\n')
        matrix_type = input('enter matrix type ','s');
        fprintf(fid, '%c\n', matrix_type(1));
    else
        fscanf(fid, '%c',1);
        matrix_type = fscanf(fid, '%c',1);
    end
    if(strncmp(matrix_type,'m',1))
        mesh(amat);
    elseif(strncmp(matrix_type,'f',1))
        foil(amat);
    else
        fprintf('matrix configuration is undefined\n')
        matrix_type = 'u';
    end
end

%=====
function mesh(amat)
% Specifies mesh matrix geometric and thermal properties
% Israel Urieli 03/31/02

global vr % regen void volume [m^3]
global ar % regen internal free flow area [m^2]
global awgr % regen internal wetted area [m^2]
global lr % regenerator effective length [m]
global dr % regen hydraulic diameter [m]
global new fid % new data file

fprintf(' stacked wire mesh matrix\n')
if(strncmp(new,'y',1))
    porosity = input('enter matrix porosity : ');

```

```

        dwire = input('enter matrix wire diameter [m] : ');
        fprintf(fid, '%.3f\n', porosity);
        fprintf(fid, '%.3e\n', dwire);
    else
        porosity = fscanf(fid,'%f',1);
        dwire = fscanf(fid,'%e',1);
    end

    ar = amat*porosity;
    vr = ar*lr;
    dr = dwire*porosity/(1 - porosity);
    awgr = 4*vr/dr;

    fprintf(' matrix porosity: %.3f\n', porosity)
    fprintf(' matrix wire diam %.2f(mm)\n', dwire*1e3)
    fprintf(' hydraulic diam %.3f(mm)\n', dr*1e3)
    fprintf(' total wetted area %.3e(sq.m)\n', awgr)
    fprintf(' regenerator length %.1f(mm)\n', lr*1e3)
    fprintf(' void volume %.2f(cc)\n', vr*1e6)
    %=====
function foil(amat)
% Specifies foil matrix geometric and thermal properties
% Israel Urieli 03/31/02

global vr % regen void volume [m^3]
global ar % regen internal free flow area [m^2]
global awgr % regen internal wetted area [m^2]
global lr % regenerator effective length [m]
global dr % regen hydraulic diameter [m]
global new fid % new data file

fprintf(' wrapped foil matrix\n')
if(strncmp(new,'y',1))
    fl = input('enter unrolled length of foil [m] : ');
    th = input('enter foil thickness [m] : ');
    fprintf(fid, '%.3f\n', fl);
    fprintf(fid, '%.3e\n', th);
else
    fl = fscanf(fid,'%f',1);
    th = fscanf(fid,'%e',1);
end

am = th*fl;
ar = amat - am;
vr = ar*lr;
awgr = 2*lr*fl;
dr = 4*vr/awgr;
porosity = ar/amat;

fprintf(' unrolled foil length: %.3f(m)\n', fl)
fprintf(' foil thickness %.3f(mm)\n',th*1e3)
fprintf(' hydraulic diam %.3f(mm)\n', dr*1e3)
fprintf(' total wetted area %f(sq.m)\n', awgr)
fprintf(' void volume %.2f(cc)\n', vr*1e6)
fprintf(' porosity %.3f\n', porosity)

%=====

```

gas.m

```

function gas
% specifies the working gas properties (he, h2, air)
% Israel Urieli 4/20/02

global rgas % gas constant [J/kg.K]
global cp % specific heat capacity at constant pressure [J/kg.K]
global cv % specific heat capacity at constant volume [J/kg.K]
global gama % ratio: cp/cv
global mu0 % dynamic viscosity at reference temp t0 [kg.m/s]
global t0 t_suth % reference temperature [K], Sutherland constant [K]
global prandtl % Prandtl number
global new fid % new data file

gas_type = 'un';
while(strncmp(gas_type,'un',2))
    if(strncmp(new,'y',1))
        fprintf('Available gas types are:\n');
        fprintf('  hydrogen\n');
        fprintf('  helium\n');
        fprintf('  air\n');
        gas_type = input('enter gas type: ','s');
        gas_type = [gas_type(1), gas_type(2)];
        fprintf(fid, '%s\n', gas_type);
    else
        fscanf(fid, '%c',1); % bypass the previous newline
character
        gas_type = fscanf(fid, '%c',2);
    end
    if(strncmp(gas_type,'hy',2))
        fprintf('gas type is hydrogen\n')
        gama = 1.4;
        rgas = 4157.2;
        mu0 = 8.35e-6;
        t_suth = 84.4;
    elseif(strncmp(gas_type,'he',2))
        fprintf('gas type is helium\n')
        gama = 1.67;
        rgas = 2078.6
        mu0 = 18.85e-6 ;
        t_suth = 80.0;
    elseif(strncmp(gas_type,'ai',2))
        fprintf('gas type is air\n')
        gama = 1.4;
        rgas = 287.0;
        mu0 = 17.08e-6; ; % < Organ 17.08e-6
        t_suth = 112.0;
    else
        fprintf('gas type is undefined\n')
        gas_type = 'un';
    end
end
cv = rgas/(gama - 1);
cp = gama*cv;
t0 = 273;
prandtl = 0.71;

```

operat.m

```

function operat(temp_h,temp_k,freq_h)
% Determine operating parameters and do Schmidt analysis
% Israel Urieli 4/20/02

global pmean % mean (charge) pressure [Pa]
global tk tr th % cooler, regenerator, heater temperatures [K]
global freq omega % cyce frequency [herz], [rads/s]
global new fid % new data file
global temp_h temp_k freq_h
if(strncmp(new,'y',1))
    pmean = input('enter mean pressure (Pa) : ');
    tk = input('enter cold sink temperature (K) : ');
    th = input('enter hot source temperature (K) : ');
    freq = input('enter operating frequency (herz) : ');
    fprintf(fid, '%.1f\n', pmean);
    fprintf(fid, '%.1f\n', tk);
    fprintf(fid, '%.1f\n', th);
    fprintf(fid, '%.1f\n', freq);
else
    pmean = fscanf(fid,'%f',1);
    tk = fscanf(fid,'%f',1);
    th = fscanf(fid,'%f',1);
    freq = fscanf(fid,'%f',1);
end
freq = freq_h;
fprintf(' Temperature ***ggggg***** (K):
%.1f\n',temp_h);
th=temp_h;
tk=temp_k;
tkk= temp_k+20;
tk=419;
tr = (922-tk)/log(922/tk);
omega = 2*pi*freq;
fprintf('operating parameters:\n');
fprintf(' mean pressure (kPa): %.3f\n',pmean*1e-3);
fprintf(' cold sink temperature (K): %.1f\n',tk);
fprintf(' hot source temperature (K): %.1f\n',th);
fprintf(' effective regenerator temperature (K): %.1f\n',tr);
fprintf(' operating frequency (herz): %.1f\n',freq);

Schmidt; % Do Schmidt analysis
%=====
function Schmidt
% Schmidt anlysis
% Israel Urieli 3/31/02

global mgas % total mass of gas in engine [kg]
global pmean % mean (charge) pressure [Pa]
global tk tr th % cooler, regen, heater temperatures [K]
global freq omega % cycle frequency [herz], [rads/s]
global vclc vcle % compression,expansion clearence vols [m^3]
global vswc vswe % compression, expansion swept volumes [m^3]
global alpha % phase angle advance of expansion space [radians]
global vk vr vh % cooler, regenerator, heater volumes [m^3]

```

```

global rgas % gas constant [J/kg.K]
trise=10;

% Schmidt analysis
c = (((vswe/th)^2 + (vswc/tk)^2 +
2*(vswe/th)*(vswc/tk)*cos(alpha))^0.5)/2;
s = (vswc/2 + vclc + vk)/tk+ vr/tr + (vswe/2 + vcle + vh)/th;
b = c/s;
sqrtb = (1 - b^2)^0.5;
bf = (1 - 1/sqrtb);
beta = atan(vswe*sin(alpha)/th/(vswe*cos(alpha)/th + vswc/tk));
fprintf(' pressure phase angle beta  %.1f(degrees)\n',beta*180/pi)
% total mass of working gas in engine
mgas=pmean*s*sqrtb/rgas

format long e
fprintf(' total mass of gas: %.8f(kg)\n',mgas)
fprintf(' total mass of gas:  %.8f(gm)\n',mgas*1e3)

% work output
wc = (pi*vswc*mgas*rgas*sin(beta)*bf/c);
we = (pi*vswe*mgas*rgas*sin(beta - alpha)*bf/c);
w = (wc + we);
power = w*freq;
eff = w/we; % qe = we
% Printout Schmidt analysis results
fprintf('===== Schmidt analysis =====\n')
fprintf(' Work(joules) %.3e,  Power(watts) %.3e\n', w,power);
fprintf(' Qexp(joules) %.3e,  Qcom(joules) %.3e\n', we,wc);
fprintf(' indicated efficiency %.3f\n', eff);
fprintf('===== \n')
Plot Schmidt analysis pv and p-theta diagrams
fprintf('Do you want Schmidt analysis plots\n');
choice = input('y)es or n)o: ','s');
if(strncmp(choice,'y',1))
    plotpv
end
Plot Alan Organ's particle mass distribution in Natural Coordinates
fprintf('Do you want particle mass distribution plot\n');
choice = input('y)es or n)o: ','s');
if(strncmp(choice,'y',1))
    plotmass
end

```

simple.m

```

function [var,dvar] = simple(temp_h)
% simple analysis - including heat transfer and pressure drop effects
% Israel Urieli, 7/22/2002 (modified 12/3/2003 for temp plots)
% Returned values:
% var(22,37) array of variable values every 10 degrees (0 - 360)
% dvar(16,37) array of derivatives every 10 degrees (0 - 360)
% Row indices of the var, dvar arrays:
TC = 1; % Compression space temperature [K]
TE = 2; % Expansion space temperature [K]
QK = 3; % Heat transferred to the cooler [J]
QR = 4; % Heat transferred to the regenerator [J]

```

```

QH = 5; % Heat transferred to the heater [J]
WC = 6; % Work done by the compression space [J]
WE = 7; % Work done by the expansion space [J]
W = 8; % Total work done (WC + WE) [J]
P = 9; % Pressure [Pa]
VC = 10; % Compression space volume [m^3]
VE = 11; % Expansion space volume [m^3]
MC = 12; % Mass of gas in the compression space [kg]
MK = 13; % Mass of gas in the cooler [kg]
MR = 14; % Mass of gas in the regenerator [kg]
MH = 15; % Mass of gas in the heater [kg]
ME = 16; % Mass of gas in the expansion space [kg]
TCK = 17; % Conditional temperature compression space / cooler [K]
THE = 18; % Conditional temperature heater / expansion space [K]
GACK = 19; % Conditional mass flow compression space / cooler [kg/rad]
GAKR = 20; % Conditional mass flow cooler / regenerator [kg/rad]
GARH = 21; % Conditional mass flow regenerator / heater [kg/rad]
GAHE = 22; % Conditional mass flow heater / expansion space [kg/rad]
ROWV = 22; % number of rows in the var matrix
ROWD = 16; % number of rows in the dvar matrix
COL = 37; % number of columns in the matrices (every 10 degrees)
%=====

global freq % cycle frequency [herz]
global tk tr th % cooler, regenerator, heater temperatures [K]
global cqwr % regenerator housing thermal conductance [W/K]
global new fish res% new data file
global temp_h
global pmean
reavg =0;
%th= temp_h;
twk = tk; % Cooler wall temp - equal to initial cooler gas temp
twh = th; % Heater wall temp - equal to initial heater gas temp
epsilon = 1;% allowable temperature error bound for cyclic convergence
terror = 10*epsilon; % Initial temperature error (to enter loop)

while (terror>epsilon)
    [var,dvar] = adiab;
    tgh = hotsim(var,twh); % new heater gas temperature
    tgk = kolsim(var,twk); % new cooler gas temperature
    terror = abs(th - tgh) + abs(tk - tgk);
    th = tgh;
    tk = tgk;
    tr = (th-tk)/log(th/tk);
end

fprintf(' converged heater and cooler mean temperatures =====\n');
fprintf('heater wall/gas temperatures: Twh = %.1f[K], Th =
%.1f[K]\n',twh,th);
fprintf('cooler wall/gas temperatures: Twk = %.1f[K], Tk =
%.1f[K]\n',twk,tk);
% Print out ideal adiabatic analysis results
eff = var(W,COL)/var(QH,COL);% engine thermal efficiency
Qkpower = var(QK,COL)*freq; % Heat transferred to the cooler (W)
Qrpower = var(QR,COL)*freq; % Heat transferred to the regenerator (W)
Qhpower = var(QH,COL)*freq; % Heat transferred to the heater (W)
Wpower = var(W,COL)*freq; % Total power output (W)
fprintf(' Work per cycle: %d\n',var(W,COL));

```

```

fprintf('==== ideal adiabatic analysis results =====\n');
fprintf(' Heat transferred to the cooler: %.2f[W]\n', Qkpower);
fprintf(' Net heat transferred to the regenerator: %.2f[W]\n',
Qrpower);
fprintf(' Heat transferred to the heater: %.2f[W]\n', Qhpower);
fprintf(' Total power output: %.2f[W]\n', Wpower);
fprintf(' Thermal efficiency: %.1f[%%]\n', eff*100);
fprintf('=====\n');

fprintf('==== Regenerator simple analysis =====\n');
[reavg,u,mu,qrloss] = regsim(var);
fprintf(' Regenerator net enthalpy loss: %.1f[W]\n', qrloss*freq);
qwrl = cqwr*(twh - twk)/freq;
fprintf(' Regenerator wall heat leakage: %.1f[W]\n', qwrl*freq);

fprintf('==== pressure drop simple analysis =====\n');
res = fopen('p_drop.csv','a');
[rho,reynold,dwork]=worksim(var,dvar,res);
fprintf(' Pressure drop available work loss: %.1f[W]\n', dwork*freq);
actWpower = Wpower - dwork*freq;
actQhpower = Qhpower + qrloss*freq + qwrl*freq;
acteff = actWpower/actQhpower;
fprintf(' Actual power from simple analysis: %.1f[W]\n', actWpower);
fprintf(' Actual heat power in from simple analysis: %.1f[W]\n',
actQhpower);
fprintf(' Actual efficiency from simple analysis: %.1f[%%]\n',
acteff*100);

format short g;

fprintf(res, '%0.5d,%0.5d,%0.5d,%0.2d,%0.2d,%0.2d,%0.2d,%0.2d,%0.2d
\n', pmean,Wpower,actWpower,actQhpower,th,twh,tk,twk,(dwork*freq));

```

adiab.m

```

function [var,dvar] = adiab
% ideal adiabatic model simulation
% Israel Urieli, 7/6/2002
% Returned values:
%   var(22,37) array of variable values every 10 degrees (0 - 360)
%   dvar(16,37) array of derivatives every 10 degrees (0 - 360)

global tk th % cooler, heater temperatures [K]

%Row indices of the var, dvar matrices, and the y,dy variable vectors:
TC = 1; % Compression space temperature (K)
TE = 2; % Expansion space temperature (K)
QK = 3; % Heat transferred to the cooler (J)
QR = 4; % Heat transferred to the regenerator (J)
QH = 5; % Heat transferred to the heater (J)
WC = 6; % Work done by the compression space (J)
WE = 7; % Work done by the expansion space (J)
W = 8; % Total work done (WC + WE) (J)
P = 9; % Pressure (Pa)
VC = 10; % Compression space volume (m^3)
VE = 11; % Expansion space volume (m^3)

```

```

MC = 12; % Mass of gas in the compression space (kg)
MK = 13; % Mass of gas in the cooler (kg)
MR = 14; % Mass of gas in the regenerator (kg)
MH = 15; % Mass of gas in the heater (kg)
ME = 16; % Mass of gas in the expansion space (kg)
TCK = 17; % Conditional temperature compression space / cooler (K)
THE = 18; % Conditional temperature heater / expansion space (K)
GACK = 19; % Conditional mass flow compression space / cooler (kg/rad)
GAKR = 20; % Conditional mass flow cooler / regenerator (kg/rad)
GARH = 21; % Conditional mass flow regenerator / heater (kg/rad)
GAHE = 22; % Conditional mass flow heater / expansion space (kg/rad)
ROWV = 22; % number of rows in the var matrix
ROWD = 16; % number of rows in the dvar matrix
COL = 37; % number of columns in the matrices (every 10 degrees)

fprintf('=====Ideal Adiabatic Analysis=====\n')
fprintf('Cooler Tk = %.1f[K], Heater Th = %.1f[K]\n', tk, th);
epsilon = 3; % Allowable error in temperature (K)
max_iteration = 20; % Maximum number of iterations to convergence
ninc = 360; % number of integration increments (every degree)
step = ninc/36; % for saving values in var, dvar matrices
dtheta = 2.0*pi/ninc; % integration increment (radians)
% Initial conditions:
y(THE) = th;
y(TCK) = tk;
y(TE) = th;
y(TC) = tk;
iter = 0;
error = 10*epsilon; % Initial error to enter the loop
% Iteration loop to cyclic convergence
while ((error >= epsilon)&(iter < max_iteration))
% cyclic initial conditions
    tc0 = y(TC);
    te0 = y(TE);
    theta = 0;
    y(QK) = 0;
    y(QR) = 0;
    y(QH) = 0;
    y(WC) = 0;
    y(WE) = 0;
    y(W) = 0;
    fprintf('iteration %d: Tc=%.1f[K],Te= %.1f[K]\n',iter,y(TC),y(TE))
    for(i = 1:1:ninc)
        [theta,y,dy] = rk4('dadiab',7,theta,dtheta,y);
    end
    error = abs(tc0 - y(TC)) + abs(te0 - y(TE));
    iter = iter + 1;
end

if (iter >= max_iteration)
    fprintf('No convergence within %d iteration\n',max_iteration)
end

% Initial var and dvar matrix
var = zeros(22,37);
dvar = zeros(16,37);

% a final cycle, to fill the var, dvar matrices

```

```

theta=0;
y(QK)=0;
y(QR)=0;
y(QH)=0;
y(WC)=0;
y(WE)=0;
y(W)=0;
[var,dvar] = filmatrix(1,y,dy,var,dvar);
for(i = 2:1:COL)
    for(j = 1:1:step)
        [theta,y,dy] = rk4('dadiab',7,theta,dtheta,y);
    end
    [var,dvar] = filmatrix(i,y,dy,var,dvar);
end

var_tran=var';
dvar_tran=dvar';

```

dadiab.m

```

function [y,dy] = dadiab(theta,y)
% Evaluate ideal adiabatic model derivatives
% Israel Urieli, 7/6/2002
% Arguments:  theta - current cycle angle [radians]
%             y(22) - vector of current variable values
% Returned values:
%             y(22) - updated vector of current variables
%             dy(16) vector of current derivatives
% Function invoked : volume.m

% global variables used from "define" functions
global vk % cooler void volume [m^3]
global vr % regen void volume [m^3]
global vh % heater void volume [m^3]
global rgas % gas constant [J/kg.K]
global cp % specific heat capacity at constant pressure [J/kg.K]
global cv % specific heat capacity at constant volume [J/kg.K]
global gama % ratio: cp/cv
global mgas % total mass of gas in engine [kg]
global tk tr th % cooler, regen, heater temperatures [K]

% Indices of the y, dy vectors:
TC = 1; % Compression space temperature (K)
TE = 2; % Expansion space temperature (K)
QK = 3; % Heat transferred to the cooler (J)
QR = 4; % Heat transferred to the regenerator (J)
QH = 5; % Heat transferred to the heater (J)
WC = 6; % Work done by the compression space (J)
WE = 7; % Work done by the expansion space (J)
W = 8; % Total work done (WC + WE) (J)
P = 9; % Pressure (Pa)
VC = 10; % Compression space volume (m^3)
VE = 11; % Expansion space volume (m^3)
MC = 12; % Mass of gas in the compression space (kg)
MK = 13; % Mass of gas in the cooler (kg)
MR = 14; % Mass of gas in the regenerator (kg)
MH = 15; % Mass of gas in the heater (kg)

```

```

ME = 16; % Mass of gas in the expansion space (kg)
TCK = 17; % Conditional temperature compression space / cooler (K)
THE = 18; % Conditional temperature heater / expansion space (K)
GACK = 19; % Conditional mass flow compression space / cooler (kg/rad)
GAKR = 20; % Conditional mass flow cooler / regenerator (kg/rad)
GARH = 21; % Conditional mass flow regenerator / heater (kg/rad)
GAHE = 22; % Conditional mass flow heater / expansion space (kg/rad)
%=====
% Volume and volume derivatives:
[y(VC),y(VE),dy(VC),dy(VE)] = volume(theta);

% Pressure and pressure derivatives:
vot = vk/tk + vr/tr + vh/th;
y(P) = (mgas*rgas/(y(VC)/y(TC) + vot + y(VE)/y(TE)));
top = -y(P)*(dy(VC)/y(TCK) + dy(VE)/y(TE));
bottom = (y(VC)/(y(TCK)*gama) + vot + y(VE)/(y(TE)*gama));
dy(P) = top/bottom;

% Mass accumulations and derivatives:
y(MC) = y(P)*y(VC)/(rgas*y(TC));
y(MK) = y(P)*vk/(rgas*tk);
y(MR) = y(P)*vr/(rgas*tr);
y(MH) = y(P)*vh/(rgas*th);
y(ME) = y(P)*y(VE)/(rgas*y(TE));
dy(MC) = (y(P)*dy(VC) + y(VC)*dy(P)/gama)/(rgas*y(TCK));
dy(ME) = (y(P)*dy(VE) + y(VE)*dy(P)/gama)/(rgas*y(TE));
dpop = dy(P)/y(P);
dy(MK) = y(MK)*dpop;
dy(MR) = y(MR)*dpop;
dy(MH) = y(MH)*dpop;

% Mass flow between cells:
y(GACK) = -dy(MC);
y(GAKR) = y(GACK) - dy(MK);
y(GAHE) = dy(ME);
y(GARH) = y(GAHE) + dy(MH);

% Conditional temperatures between cells:
y(TCK) = tk;
if(y(GACK)>0)
    y(TCK) = y(TC);
end
y(TE) = y(TE);
if(y(GAHE)>0)
    y(TE) = th;
end

% 7 derivatives to be integrated by rk4:
% Working space temperatures:
dy(TC) = y(TC)*(dpop + dy(VC)/y(VC) - dy(MC)/y(MC));
dy(TE) = y(TE)*(dpop + dy(VE)/y(VE) - dy(ME)/y(ME));

% Energy:
dy(QK) = vk*dy(P)*cv/rgas - cp*(y(TCK)*y(GACK) - tk*y(GAKR));
dy(QR) = vr*dy(P)*cv/rgas - cp*(tk*y(GAKR) - th*y(GARH));
dy(QH) = vh*dy(P)*cv/rgas - cp*(th*y(GARH) - y(TE)*y(GAHE));
dy(WC) = y(P)*dy(VC);
dy(WE) = y(P)*dy(VE);

```

```

% Net work done:
dy(W) = dy(WC) + dy(WE);
y(W)=y(WC)+y(WE);

```

```

volume.m

function [vc,ve,dvc,dve] = volume(theta)
% determine working space volume variations and derivatives
% Israel Urieli, 7/6/2002
% Argument: theta - current cycle angle [radians]
% Returned values:
%   vc, ve - compression, expansion space volumes [m^3]
%   dvc, dve - compression, expansion space volume derivatives

global engine_type % sinusoidal, yoke (both alpha engines)

if(strncmp(engine_type,'s',1))
    [vc,ve,dvc,dve] = e_rhombic(theta); %GPU3
end

%=====
function [vc,ve,dvc,dve] = e_rhombic(theta)

% Argument: theta - current cycle angle [radians]
% Returned values:
%   vc, ve - compression, expansion space volumes [m^3]
%   dvc, dve - compression, expansion space volume derivatives
global vcd vhd % compression, expansion clearance vols [m^3]
global vswc vswe % compression, expansion swept volumes [m^3]
global alpha % phase angle advance of expansion space [radians]
global D L e r

D=0.0699;
r=0.01397;
e=0.02065;
L=0.04602;
d_rod=0.00953;

vclc=2.119e-005; %clearance compress volume
vcle=1.249e-005; %clearance expansion volume

b1=sqrt(L^2-(e-r)^2);
b2= sqrt((L-r)^2-e^2);
b4= sqrt((L+r)^2-e^2);
b_theta=sqrt(L^2-(e+r*cos(theta))^2);

ap=(pi/4)*(D^2-d_rod^2);% Area piston
ad=(pi/4)*D^2 ;      % Area displacer

vc_l= 2*ap*(b1-b_theta);
ve_l= ad*(b_theta-b2-r*sin(theta));

dvc=-2*ap*r*sin(theta)*(e+r*cos(theta))/b_theta ;
dve=-(dvc*ad/(2*ap))-(r*cos(theta)*ad) ;

vc=vc_l+vclc;

```

```
ve=vcle+ve_l;
```

rk4.m

```
function [x, y, dy] = rk4(deriv,n,x,dx,y)
%Classical fourth order Runge-Kutta method
%Integrates n first order differential equations
%dy(x,y) over interval x to x+dx
%Izzi Urieli - Jan 21, 2002
x0 = x;
y0 = y;
[y,dy1] = feval(deriv,x0,y);
for i = 1:n
    y(i) = y0(i) + 0.5*dx*dy1(i);
end
xm = x0 + 0.5*dx;
[y,dy2] = feval(deriv,xm,y);
for i = 1:n
    y(i) = y0(i) + 0.5*dx*dy2(i);
end
[y,dy3] = feval(deriv,xm,y);
for i = 1:n
    y(i) = y0(i) + dx*dy3(i);
end
x = x0 + dx;
[y,dy] = feval(deriv,x,y);
for i = 1:n
    dy(i) = (dy1(i) + 2*(dy2(i) + dy3(i)) + dy(i))/6;
    y(i) = y0(i) + dx*dy(i);
end
```

regsim.m

```
function [reavg,u,mu,qrloss] = regsim(var)
% Evaluate the effectiveness and performance of the regenerator
% Israel Urieli, 7/23/2002
% Arguments:
%   var(22,37) array of variable values every 10 degrees (0 - 360)
% Returned value:
%   qrloss - regenerator net enthalpy loss [J]

% Row indices of the var array:
TC = 1; % Compression space temperature [K]
TE = 2; % Expansion space temperature [K]
QK = 3; % Heat transferred to the cooler [J]
QR = 4; % Heat transferred to the regenerator [J]
QH = 5; % Heat transferred to the heater [J]
WC = 6; % Work done by the compression space [J]
WE = 7; % Work done by the expansion space [J]
W = 8; % Total work done (WC + WE) [J]
P = 9; % Pressure [Pa]
VC = 10; % Compression space volume [m^3]
VE = 11; % Expansion space volume [m^3]
MC = 12; % Mass of gas in the compression space [kg]
MK = 13; % Mass of gas in the cooler [kg]
MR = 14; % Mass of gas in the regenerator [kg]
MH = 15; % Mass of gas in the heater [kg]
```

```

ME = 16; % Mass of gas in the expansion space [kg]
TCK = 17; %Conditional temperature compression space / cooler [K]
THE = 18; %Conditional teperature heater / expansion space [K]
GACK = 19; %Conditional mass flow compression space / cooler [kg/rad]
GAKR = 20; % Conditional mass flow cooler / regenerator [kg/rad]
GARH = 21; % Conditional mass flow regenerator / heater [kg/rad]
GAHE = 22; % Conditional mass flow heater / expansion space [kg/rad]

global matrix_type % m)esh or f)oil
global ar % regen internal free flow area [m^2]
global awgr % regen internal wetted area [m^2]
global dr % regen hydraulic diameter [m]
global tr % regen temperature [K]
global omega % cycle frequency [rads/s]
global mu
global kgasr rho cp
global pmean

% Reynolds number over the cycle
for(i = 1:1:37)
    gar(i) = (var(GAKR,i) + var(GARH,i))*omega/2;
    gr = gar(i)/ar;
    [mu,kgasr,re(i)] = reynum_t(tr,gr,dr);
    %mu =mu ;
end

% average and maximum Reynolds number
sumre = 0;
remax = re(1);
for(i = 1:1:36)
    sumre = sumre + re(i);
    if(re(i) > remax)
        remax = re(i);
    end
end

reavg = sumre/36;
u= reavg*(1-0.697)*mu/(0.04006e-3*0.697);

% Stanton number, number of transfer units, regenerator effectiveness
if (strncmp(matrix_type,'m',1))
    [st,fr] = matrixfr(reavg);
elseif (strncmp(matrix_type,'f',1))
    [st,ht,fr] = foilfr(dr,mu,reavg);
end

nu=(1+0.99*(reavg*0.7)^0.66)*0.697^1.79;

rho =1*pmean/(2.0769*tr*1000) ;% kg/m3
ntu = st*awgr/(2*ar);
effect = ntu/(ntu + 1);

% Calculate qrloss
for (i=1:1:37)
    qreg(i) = var(QR,i);
end
qrmin = min(qreg);
qrmax = max(qreg);

```

```

qrloss =(1- effect)*(qrmax - qrmin);
fr_new=(151.2346/re(i))+(3.994/re(i)^0.103

% Regenerator simple analysis results:
fprintf('Average Reynolds number: %.1f\n', reavg);
fprintf('Maximum Reynolds number: %.1f\n', remax);
fprintf('Stanton number(Average Re): %.3f\n',st);
fprintf('Number of transfer units: %.1f\n',ntu);
fprintf('Friction Factor: =%d ,Friction Factor2 : =%d \n',fr,fr_new);
fprintf('Regenerator effectiveness : %.3f\n',effect);

```

worksim.m

```

function [rho,reynold,dwork] = worksim(var,dvar,res);
% call by simple line 113
% Evaluate the pressure drop available work loss [J]
% Israel Urieli, 7/23/2002
% Arguments:
%   var(22,37) array of variable values every 10 degrees (0 - 360)
%   dvar(16,37) array of derivatives every 10 degrees (0 - 360)
%   Returned value:
%   dwork - pressure drop available work loss [J]
% Row indices of the var, dvar arrays:
TC = 1; % Compression space temperature [K]
TE = 2; % Expansion space temperature [K]
QK = 3; % Heat transferred to the cooler [J]
QR = 4; % Heat transferred to the regenerator [J]
QH = 5; % Heat transferred to the heater [J]
WC = 6; % Work done by the compression space [J]
WE = 7; % Work done by the expansion space [J]
W = 8; % Total work done (WC + WE) [J]
P = 9; % Pressure [Pa]
VC = 10; % Compression space volume [m^3]
VE = 11; % Expansion space volume [m^3]
MC = 12; % Mass of gas in the compression space [kg]
MK = 13; % Mass of gas in the cooler [kg]
MR = 14; % Mass of gas in the regenerator [kg]
MH = 15; % Mass of gas in the heater [kg]
ME = 16; % Mass of gas in the expansion space [kg]
TCK = 17; %Conditional temperature compression space / cooler [K]
THE = 18; %Conditional temepature heater / expansion space [K]
GACK = 19; %Conditional mass flow compression space / cooler [kg/rad]
GAKR = 20; % Conditional mass flow cooler / regenerator [kg/rad]
GARH = 21; % Conditional mass flow regenerator / heater [kg/rad]
GAHE = 22; % Conditional mass flow heater / expansion space [kg/rad]
ROWV = 22; % number of rows in the var matrix
ROWD = 16; % number of rows in the dvar matrix
COL = 37; % number of columns in the matrices (every 10 degrees)
%=====

global tk tr th % cooler, regenerator, heater temperatures [K]
global freq omega % cycle frequency [herz], [rads/s]
global vh % heater void volume [m^3]
global ah % heater internal free flow area [m^2]
global dh % heater hydraulic diameter [m]
global lh % heater effective length [m]
global vk % cooler void volume [m^3]

```

```

global ak % cooler internal free flow area [m^2]
global dk % cooler hydraulic diameter [m]
global lk % cooler effective length [m]
global vr % regen void volume [m^3]
global ar % regen internal free flow area [m^2]
global lr % regenerator effective length [m]
global dr % regen hydraulic diameter [m]
global matrix_type % m)esh or f)oil
global pmean
global rho
global res
global rgas
dtheta = 2*pi/36;
dwork = 0; % initialise pumping work loss
reynold = 0;
porous=0.697;% porosity
T0=273.15;
fprintf(' Rgas: %.3f[K]\n', rgas);
vi=rgas*tr/(pmean);
rho=1/vi ;% kg/m3
d_regen=22.6e-3;

Ar2=(pi/4)*d_regen^2 ;
for(i = 1:1:36)
%Pressure drop cooler
    gk = (var(GACK,i) + var(GAKR,i))*omega/(2*ak);
    [mu,kgas,re(i)] = reynum(tk,gk,dk);
    [ht,fr] = pipefr(dk,mu,re(i));
    dpkol(i) = 2*fr*mu*vk*gk*lk/(var(MK,i)*dk^2);

%Pressure drop heater
    gh = (var(GARH,i) + var(GAHE,i))*omega/(2*ah);
    [mu,kgas,re(i)] = reynum(th,gh,dh);
    [ht,fr] = pipefr(dh,mu,re(i));
    dphot(i) = 2*fr*mu*vh*gh*lh./(var(MH,i)*dh^2);

%Pressure drop regen
    gr = (var(GAKR,i) + var(GARH,i))*omega/(2*ar); %originall
    [mu,kgas,re(i)] = reynum_t(tr,(gr/rho),dr);
    reynold = re(i);
    Cf2=(151.2346/re(i))+3.994/re(i)^0.103 ;

    if(strncmp(matrix_type,'m',1))
        [st,fr] = matrixfr(re(i));
    elseif (strncmp(matrix_type,'f',1))
        [st,ht,fr] = foilfr(dr,mu,re(i));
    end

a0=8*pi/4*(22.6e-3)^2;
u=gr/rho;
n=308;
L=22.6e-3;
d=0.04006e-3;

dpreg(i) = Cf2*L*(1-0.697)*rho*u^2*(u/abs(u))/(2*0.697*d);
dp(i) = dpkol(i) + dpreg(i) + dphot(i);
dwork=dwork+dtheta*dp(i)*dvar(VE,i); % pumping work [J]
pcom(i) = var(P,i);

```

```

        pexp(i) = pcom(i) + dp(i);
end

dpkol(COL) = dpkol(1);
dpreg(COL) = dpreg(1);
dphot(COL) = dphot(1);
dp(COL) = dp(1);
pcom(COL) = pcom(1);
pexp(COL) = pexp(1);

fprintf('quitting pressure plots...\n');

```

reynum.m

```

function [mu,kgas,re] = reynum(t,g,d)
% evaluate dynamic viscosity, thermal conductivity, Reynolds number
% Israel Urieli, 7/22/2002
% Arguments:
%   t - gas temperature [K]
%   g - mass flux [kg/m^2.s]
%   d - hydraulic diameter [m]
% Returned values:
%   mu - gas dynamic viscosity [kg.m/s]
%   kgas - gas thermal conductivity [W/m.K]
%   re - Reynolds number

global cp % specific heat capacity at constant pressure [J/kg.K]
global mu0 % dynamic viscosity at reference temp t0 [kg.m/s]
global t0 t_suth % reference temperature [K], Sutherland constant [K]
global prandtl % Prandtl number

mu = mu0*(t0 + t_suth)/(t + t_suth)*(t/t0)^1.5;
kgas = cp*mu/prandtl;
re = abs(g)*d/mu;
%disp(re)
% fprintf(' display Reynolds number: %.3f\n', re);
if(re < 1)
    re = 1;
end

```

reynum_t.m

```

function [mu,kgasr,re] = reynum_t(t,g,d)
% evaluate dynamic viscosity, thermal conductivity, Reynolds number
% Israel Urieli, 7/22/2002
% Arguments:
%   t - gas temperature [K]
%   g - mass flux [kg/m^2.s]
%   d - hydraulic diameter [m]
% Returned values:
%   mu - gas dynamic viscosity [kg.m/s]
%   kgas - gas thermal conductivity [W/m.K]
%   re - Reynolds number

global cp % specific heat capacity at constant pressure [J/kg.K]
global mu0 % dynamic viscosity at reference temp t0 [kg.m/s]
global t0 t_suth % reference temperature [K], Sutherland constant [K]

```

```

global prandtl

mu = mu0*(t0 + t_suth)/(t + t_suth)*(t/t0)^1.5;
kgasr = cp*mu/prandtl;
porous=0.697;
A_reg=8*(pi/4)*(22.6e-3)^2 ;
u= abs(g)/1;
    s0= 0.1265822e-003 ;
    w0= 0.0865822e-003;

re=0.04006e-3*porous*u/((1-porous)*mu) ;
if(re < 1)
    re = 1;
end

```

ง2 โปรแกรมคำนวณกำลังบ่งชี้โดยเปลี่ยนอัตราส่วน e/L และ r/L

โปรแกรมสำหรับคำนวณกำลังบ่งชี้ เมื่อเปลี่ยนแปลงอัตราส่วน e/L และ r/L ตั้งแต่ 0.1-0.8 และ 0.1-0.5 ปรับปรุงโปรแกรมเดิมจาก ง1 ให้ทำการคำนวณโดยการวนรอบเปลี่ยนค่า e/L และ r/L โดยปรับปรุงไฟล์ cycle.m และ volume.m

cycle.m

```

clc;
clear all;
global res
global temp_h temp_k
global freq_h
global e_l r_l
temp=[965 773 969];
tempk=[337 330 353];
rpm=[41.66 50 58.33];

for (j = 0.4:0.05:0.4) % (j = 0.1:0.05:0.8)
    for (k = 0.5:0.05:0.5)

        e_l =j ;%k
        r_l =k ;%j

        for(i = 1:1:3)

            if ((j+k) <= 1.00)

                temp_h=temp(i);
                temp_k=tempk(i);
                freq_h=rpm(i);
                sea(temp_h,temp_k,freq_h);
            else
                sea(0,0,0);
            end

        end

    end

end
end

```

end

volume.m

```
function [vc,ve,dvc,dve] = volume(theta) %call by dadiab line 48
% determine working space volume variations and derivatives
% Israel Urieli, 7/6/2002
% Argument:  theta - current cycle angle [radians]
% Returned values:
%   vc, ve - compression, expansion space volumes [m^3]
%   dvc, dve - compression, expansion space volume derivatives

global engine_type % s)inusoidal, y)oke (both alpha engines)
global e_l
global r_l

if(strncmp(engine_type,'s',1))
    [vc,ve,dvc,dve] = e_rhombic(theta); %GPU3
end

%=====
function [vc,ve,dvc,dve] = sinevol(theta)
% sinusoidal drive volume variations and derivatives
% Israel Urieli, 7/6/2002
% Argument:  theta - current cycle angle [radians]
% Returned values:
%   vc, ve - compression, expansion space volumes [m^3]
%   dvc, dve - compression, expansion space volume derivatives

global vclc vcle % compression,expansion clearence vols [m^3]
global vswc vswe % compression, expansion swept volumes [m^3]
global alpha % phase angle advance of expansion space [radians]

vc = vclc + 0.5*vswc*(1 + cos(theta));
ve = vcle + 0.5*vswe*(1 + cos(theta + alpha));
dvc = -0.5*vswc*sin(theta);
dve = -0.5*vswe*sin(theta + alpha);
%=====
% GPU
function [vc,ve,dvc,dve] = e_rhombic(theta,e_l,r_l)
% sinusoidal drive volume variations and derivatives
% Israel Urieli, 7/6/2002
% Argument:  theta - current cycle angle [radians]
% Returned values:
%   vc, ve - compression, expansion space volumes [m^3]
%   dvc, dve - compression, expansion space volume derivatives
global vcd vhd % compression,expansion clearence vols [m^3]
global vswc vswe % compression, expansion swept volumes [m^3]
global alpha % phase angle advance of expansion space [radians]
global D L e r
global e_l r_l

D=0.0699;
L=0.046;
d_rod=0.00953;
e = e_l*L;
r = r_l*L;
```

```
vclc=2.119e-005; %clearance compress volume
vcle=1.249e-005; %clearance expansion volume

b1=sqrt(L^2-(e-r)^2);
b2= sqrt((L-r)^2-e^2);
b4= sqrt((L+r)^2-e^2);
b_theta=sqrt(L^2-(e+r*cos(theta))^2);

ap=(pi/4)*(D^2-d_rod^2);% Area piston
ad=(pi/4)*D^2 ;          % Area displacer

vc_l= 2*ap*(b1-b_theta);
ve_l= ad*(b_theta-b2-r*sin(theta));

dvc=-2*ap*r*sin(theta)*(e+r*cos(theta))/b_theta ;
dve=-(dvc*ad/(2*ap))-(r*cos(theta)*ad) ;

vc=vclc+vc_l;
ve=vcle+ve_l;
```
